

Piezo-Thermo-Mechanical FEM analysis applied to *vibrating inertial microsensors*



Inertial microsensors at ONERA

2 kinds of inertial microsensors

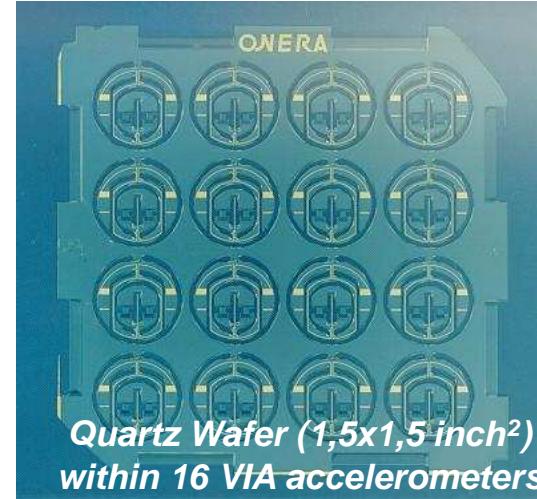
- Accelerometers VIA & DIVA
- Rate gyro VIG

Application

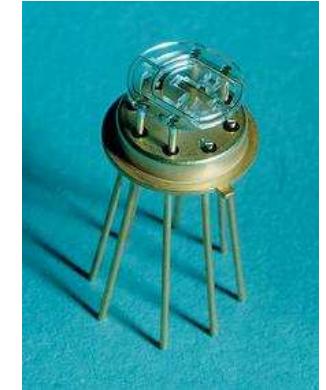
- Inertial Measurement Unit
 - 3 accelerometers
 - 3 gyros

Quartz

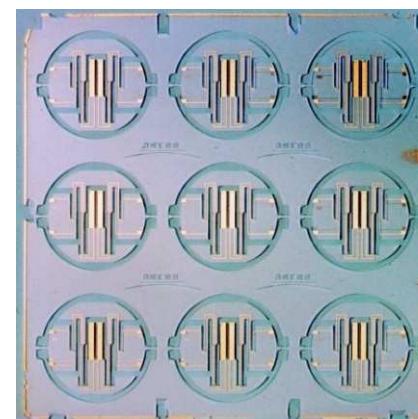
- Piezoelectricity
- Thermal stability



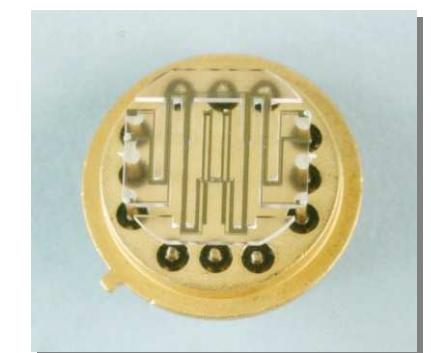
Quartz Wafer ($1,5 \times 1,5 \text{ inch}^2$)
within 16 VIA accelerometers



VIA Accelerometer



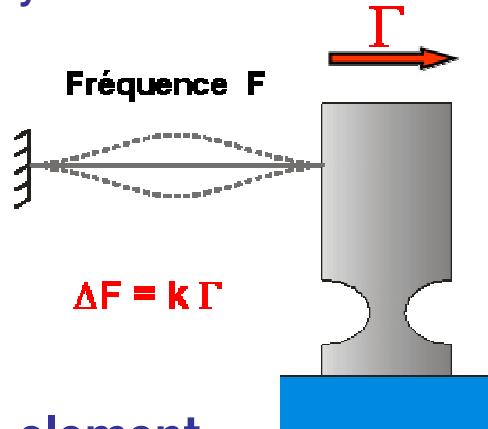
Quartz Wafer ($1,5 \times 1,5 \text{ inch}^2$)
within 9 VIG Gyros



VIG Gyro

Accelerometers VIA & DIVA

- ❑ Vibrating Beam Accelerometer
 - ❑ Frequency shift due to axial stresses

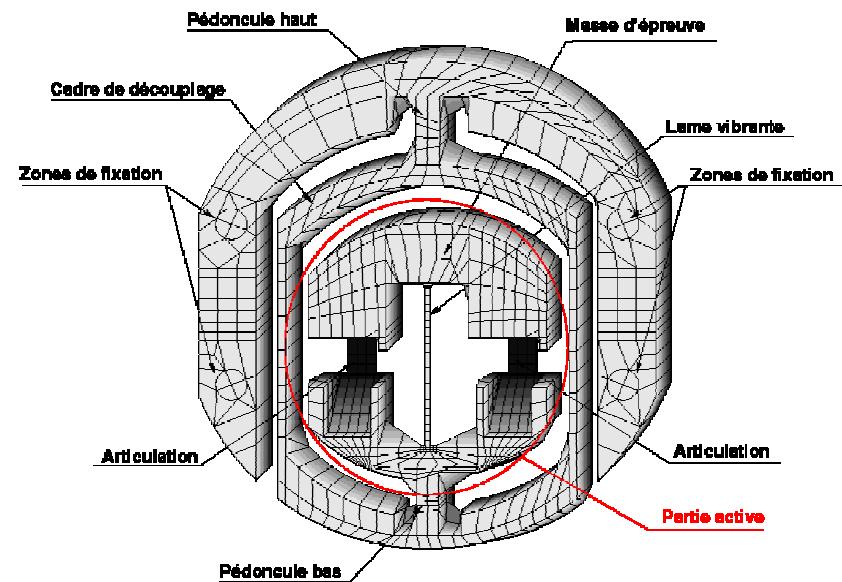


- ☐ Sensitive element
 - ☐ Beam : $60 \mu\text{m} \times 30 \mu\text{m} \times 2.2 \text{ mm}$
 - ☐ Proof mass : 5 mg
 - ☐ Sensitive to orthogonal acceleration

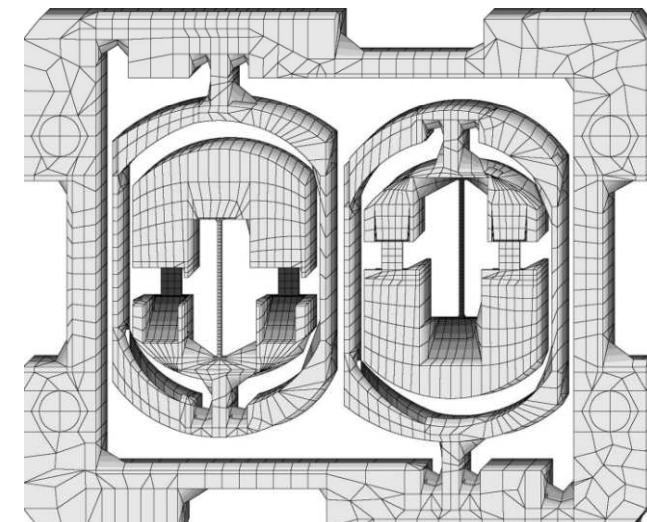
- Detection system
 - Piezoelectric excitation
 - Electronic oscillator

- ❑ Monolithic differential accelerometer
 - ❑ DIVA

VIA

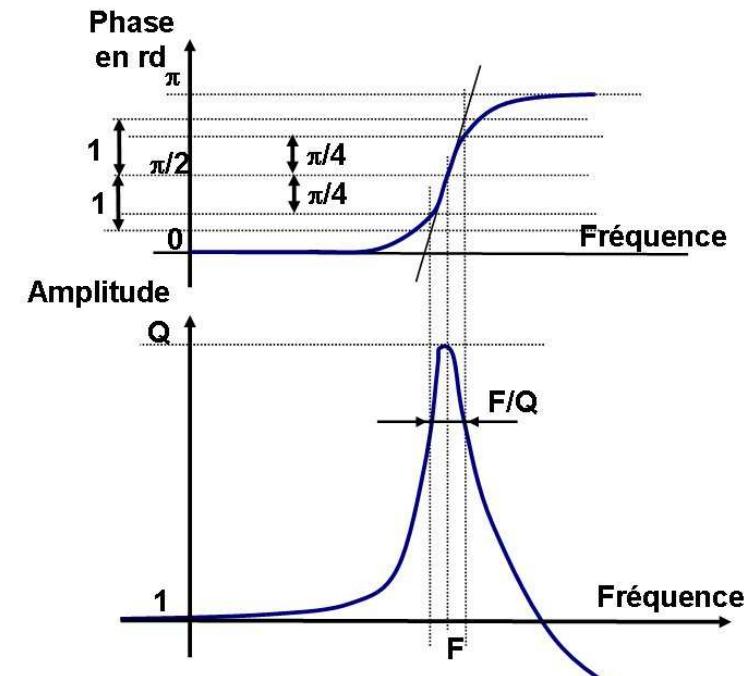


DIVA



High-Q resonators

- Oscillator accuracy
 - High Q-factors required
- $$\Delta F = \frac{F}{2.Q} \Delta \varphi$$
- Energy dissipation
 - Gas damping (neglected)
 - Vacuum (10^{-2} mbar)
 - Thermoelastic damping
 - Clamp losses
- FEM Analysis
 - Multiphysic approach required



$$Q = 2\pi \cdot \frac{W_{stockée}}{W_{dissipée}} \Rightarrow \frac{1}{Q} = \frac{1}{Q_i} + \frac{1}{Q_s} + \frac{1}{Q_d}$$

Multiphysic FEM

□ Needs

□ Mechanical behavior

- Anisotropic material
- 3D structure
- Quality Factor prediction
 - Thermoelastic damping

□ Resonator behavior

- Electrical parameters
 - Piezoelectric coupling

□ Sensor

- Scale factor

□ Multiphysic approach

□ Piezo-thermo-elastic FEM

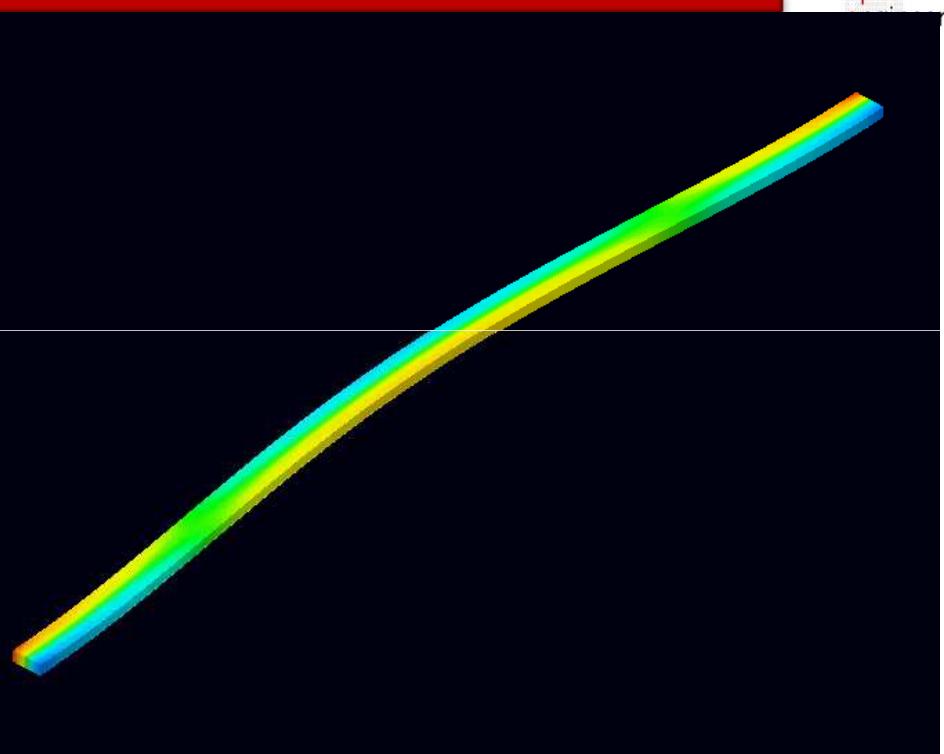
- OOFELIE (Open Engineering)
- Samcef Field (Samtech)

$$\left\{ \begin{array}{l} T_i = C_{ij}^E \cdot (S_j - \alpha_i \theta) - e_{ki}^t \cdot E_k \\ D_i = \varepsilon_{ij}^S \cdot E_j + e_{ijk} \cdot S_{jk} + p_i \theta \\ \sigma = (C_{ij}^E \alpha_i)^t S_i + p_i^t E_i + \frac{C_p}{T_0} \theta \end{array} \right.$$

$$\left\{ \begin{array}{l} \rho \frac{\partial^2 u_i}{\partial t^2} = \frac{\partial T_{ik}}{\partial x_k} + f \\ \frac{\partial D_i}{\partial x_i} = \rho_e \\ (T_0 + \theta) \dot{\sigma} = \lambda \Delta \theta + u_{th} \end{array} \right.$$

Thermoelastic Damping

- Bending mode**
 - Compression -> heating
 - Extension -> cooling
- Irreversible heat flow**
 - Energy dissipation
 - Damping
- Limitation of analytical model**
 - Anisotropic material
 - Complex 3D structure
- Modeling using Oofelie**
 - Harmonic response analysis
 - Influence of piezoelectricity
 - Good agreement with experimental results



$$Q_{\text{thermo}} = \frac{\rho \cdot C}{\alpha^2 \cdot T \cdot E} \cdot \frac{F_o^2 + F^2}{F_o F} \quad \text{avec } F_o = \frac{\pi \cdot D}{2 \cdot e^2}$$

	Q factor
Zener theory	16 580
Oofelie : thermo-elastic	13 700
Oofelie : piezo-thermo-elastic	13 090
Experimental characterisation	~13 000

S. Lepage et al., CANEUS 2006, Toulouse, France

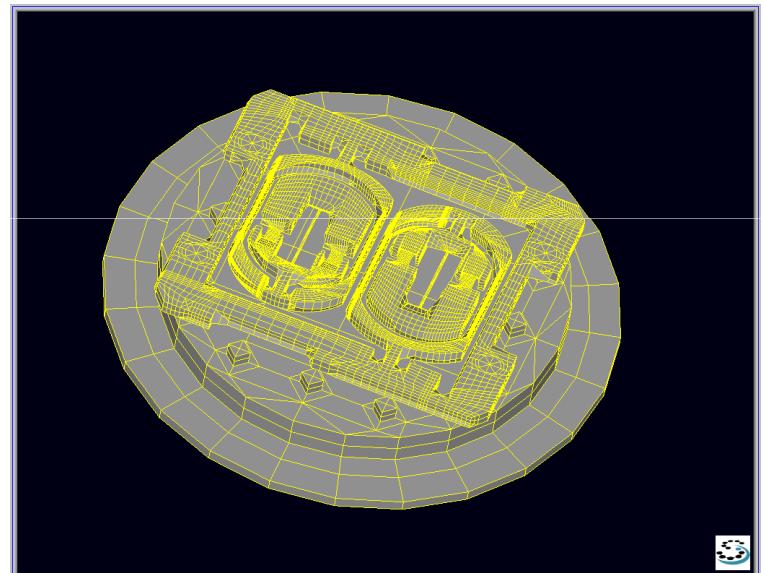
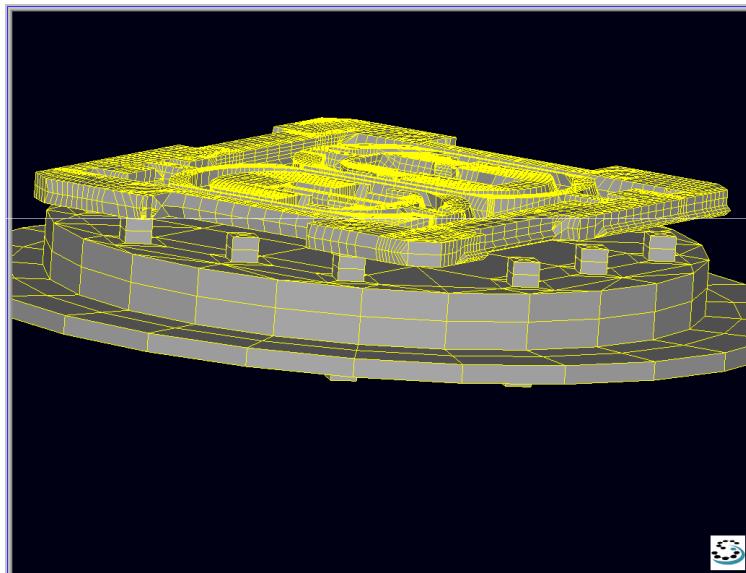
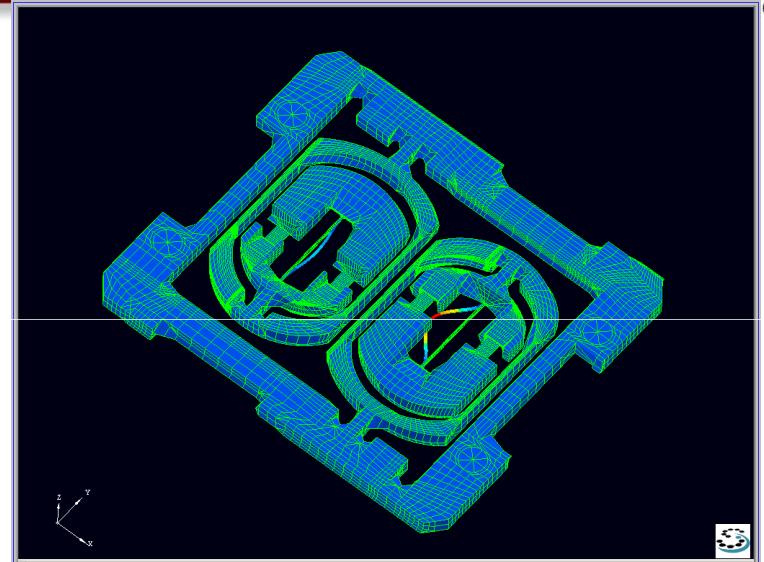
Insulating Frame

□ Goal

- Limit energy losses through mounting parts
- Preserve resonance quality
- Protect resonance frequency from thermal stresses

□ FEM Analysis

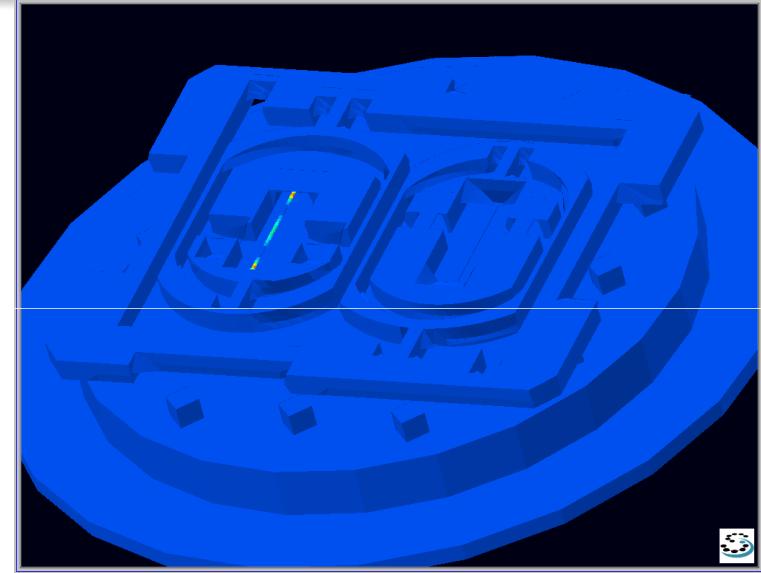
- Model quartz structure + TO8 base



Insulating Frame

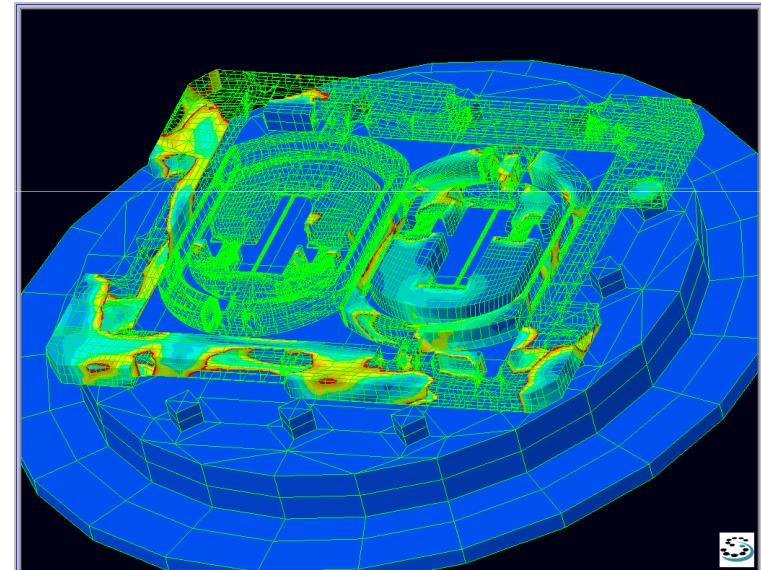
Prediction of the frame efficiency

- Modal Analysis
- Evaluation of the strain energy dissipated in the base



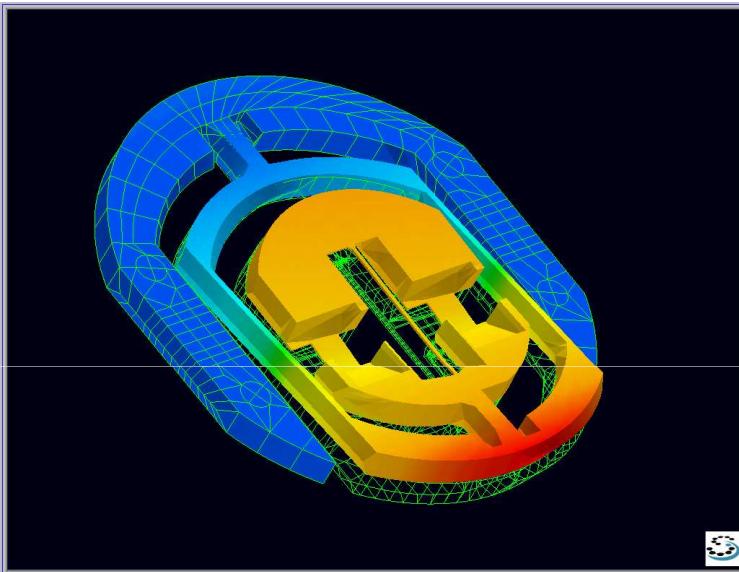
Less than 10^{-8} of total vibrating energy in mounting parts

- $Q_{\text{decoupling}} > 10^8$
- Compatible with thermoelastic damping ($Q_{\text{th}} = 13000$)



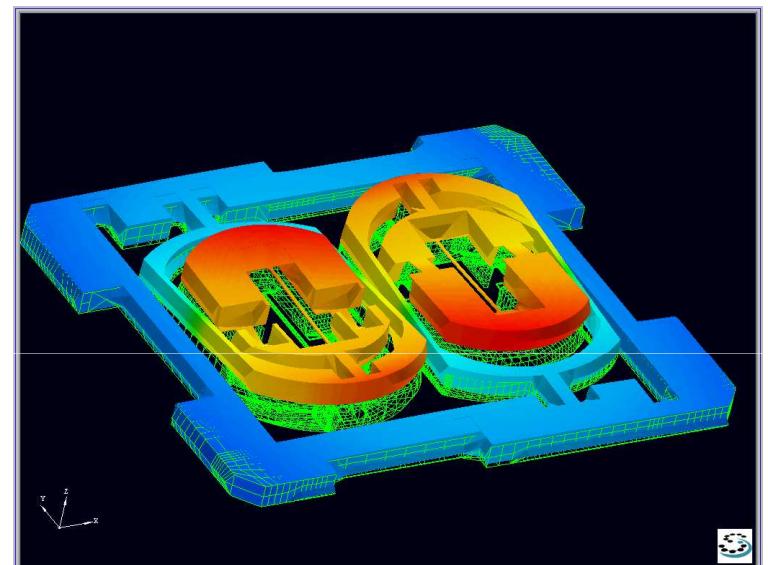
Scale Factor Estimation

- Stress generated by static acceleration
- Modal analysis with static pre-stress
- Evaluation of the frequency shift due to acceleration



Numeric scale factor : 12.6 Hz/g

Experimental S.F. : ~ 12.5 Hz/g



Numeric scale factor : 31.9 Hz/g

Experimental S.F. : ~ 30.5 Hz/g

Electric behavior (1/2)

❑ Equivalent electric model

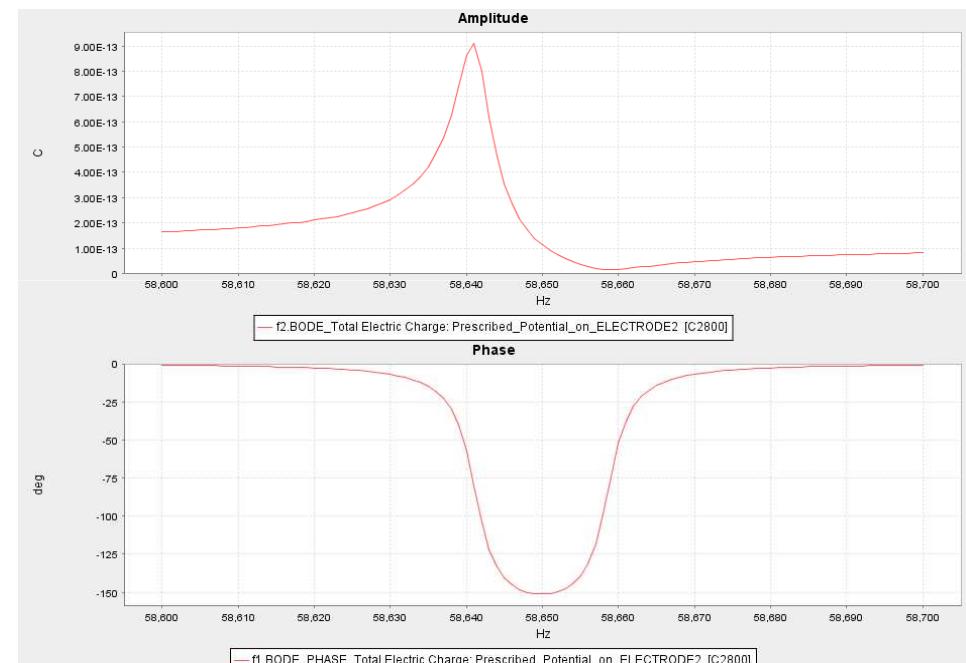
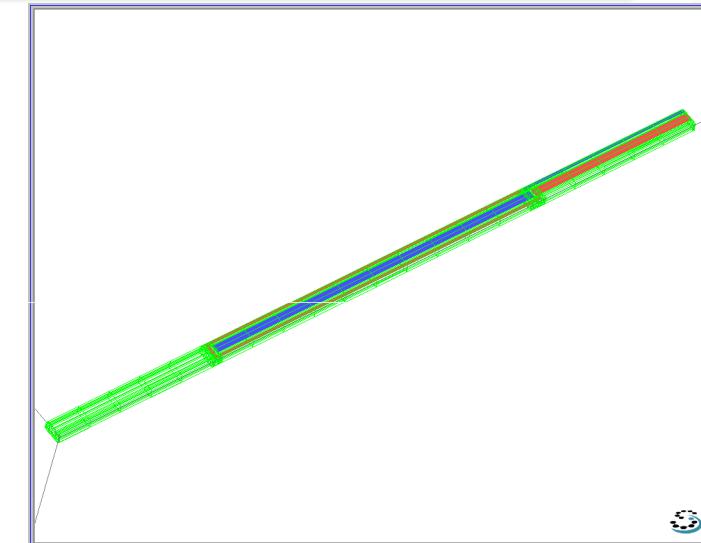
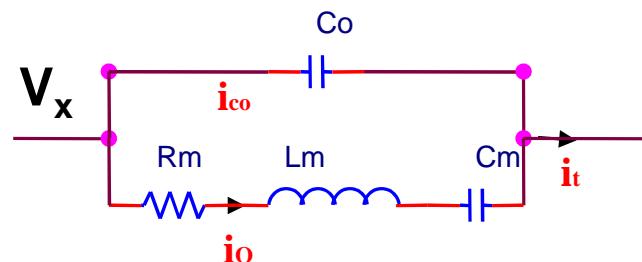
- ❑ C_0 : Capacitance
- ❑ R_m , L_m , C_m : motionnal parameters.

❑ Influence on electronic oscillateur

❑ Piezoelectric FEM analysis

- ❑ Electric response of the transducer
- ❑ Motionnal parameters
 - ❑ $C_0 \# 1 \text{ pF}$
 - ❑ $R_m \# 3 \text{ M}\Omega$
 - ❑ Très bon accord avec l'expérience

❑ Phase shift induced by C_0

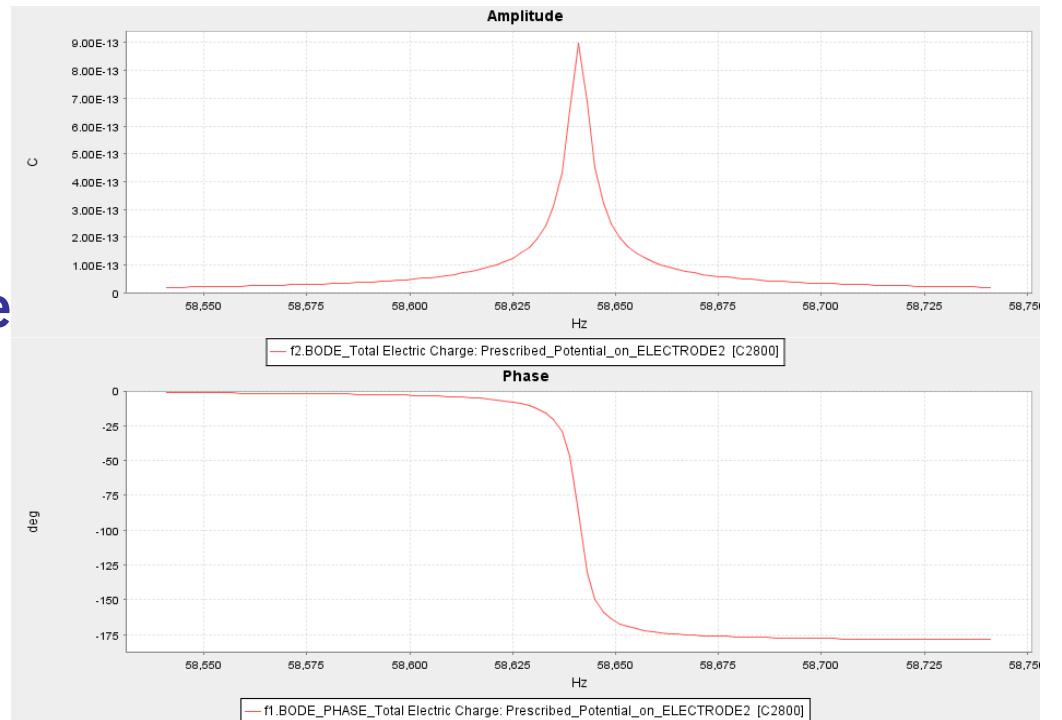
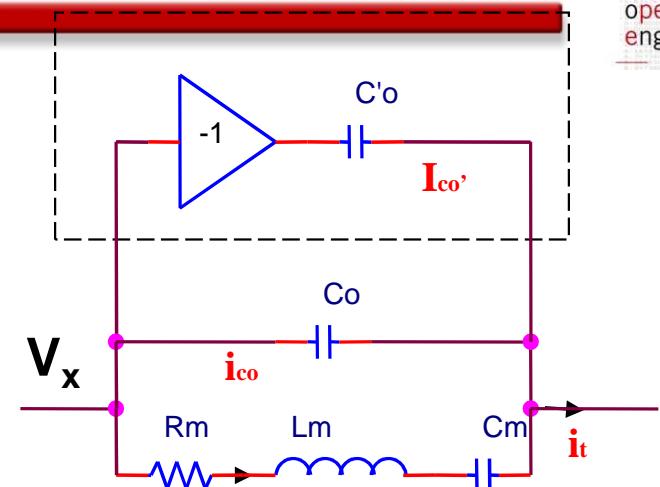


Electric behavior (2/2)

- Influence of external electric impedance
 - Inter electrode capacitance cancellation

- Impact of the electronic circuit on the transducer behavior
 - Phase shift cancelled
 - Same quality factor

- Better response of the transduce



DIVA : Lock-in phenomena

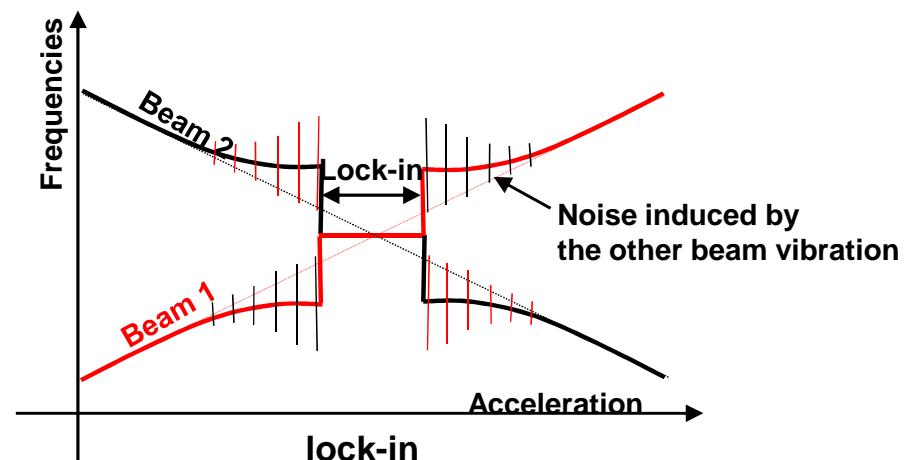
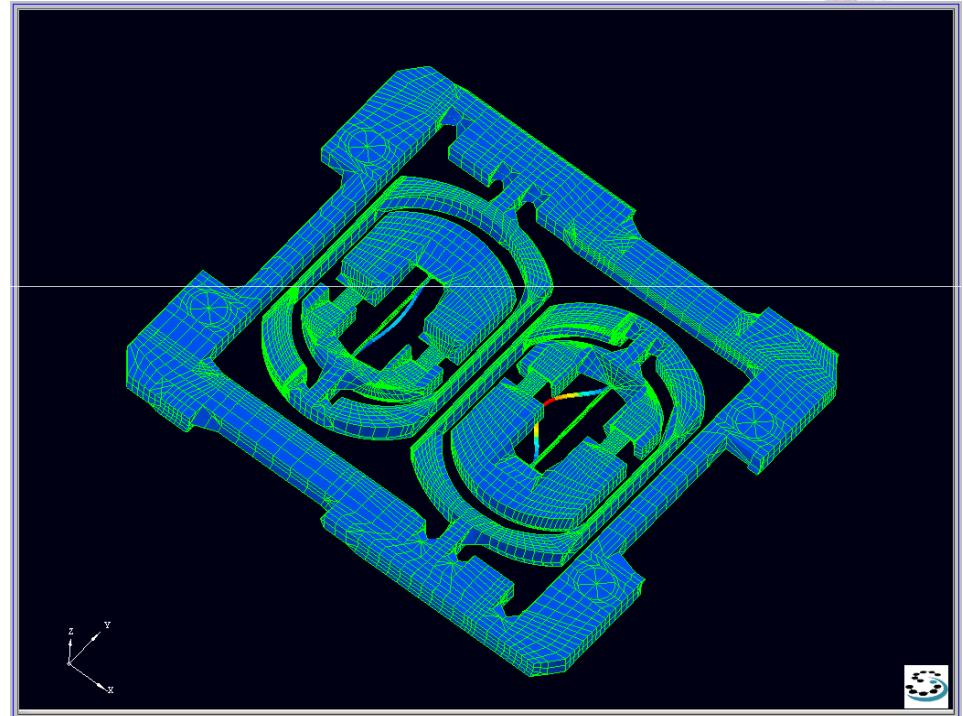
❑ Lock-in

- ❑ Mechanical coupling between resonators
- ❑ Same resonance frequencies
- ❑ Blind zone

❑ Specific optimization by FEM

- ❑ Decoupling frame optimization
- ❑ Reduce vibrating energy transfer between resonators

❑ Reduction of the blind zone to 1 mg

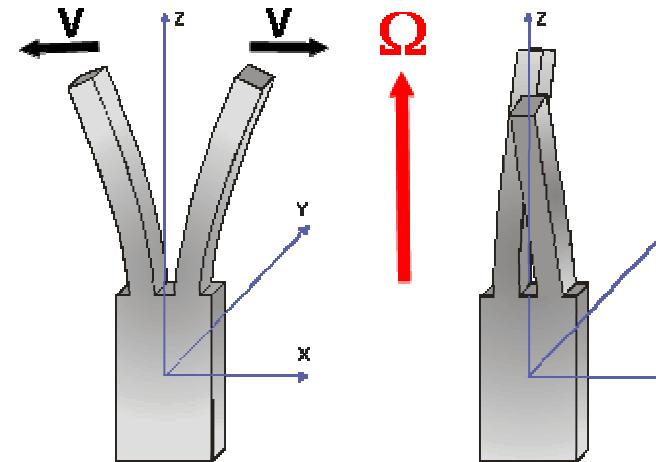


Gyro VIG

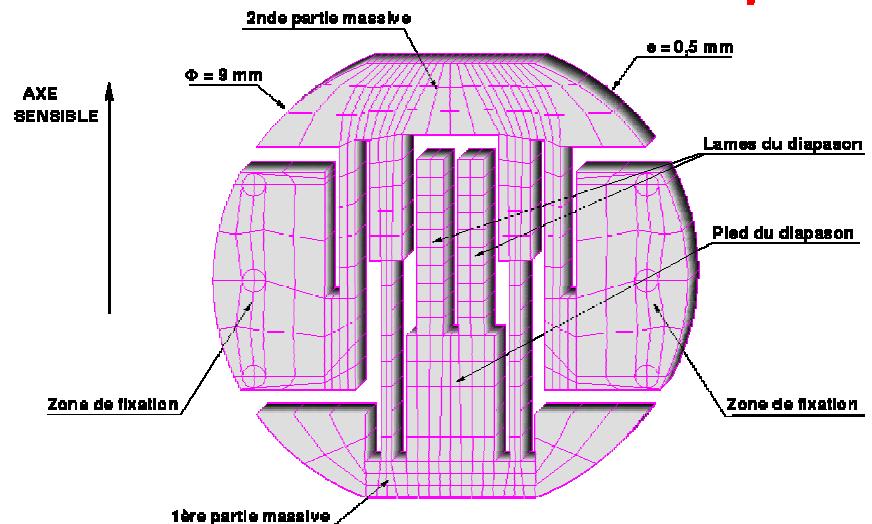
INT-PIRES
8-3-2300E-03
2-15335B0E-03
4-940170E-03
1-007500E-02
-1-271910E-03
6-077220E-03
-2-175770E-02
5-496530E+00
1-177160E-02
2-175770E-03
5-496530E-03
6-271910E-03
6-077220E-03
1-000000E+00
1-000000E+00
3-1100010E-03
4-007500E-03
1-000000E+00
1-000000E-03
2-214215E-03
4-007500E-03
-1-033717E-03
0-000000E+00
0-000000E+00
-1-032491E-03
0-000000E+00
0-000000E+00
1-000000E-03

Coriolis Vibrating Gyro

- ❑ Sensitive element: tuning fork
 - ❑ $500 \mu\text{m} * 500 \mu\text{m} * 2 \text{ mm}$
- ❑ Driving mode : in-plane bending resonance ($\sim 35 \text{ kHz}$)
- ❑ Sensing mode : orthogonal bending mode induced by coriolis acceleration
- ❑ Angular rate measured by the amplitude of the sensing mode



$$\vec{\Gamma}_c = 2 \vec{\Omega} \wedge \vec{v} \Rightarrow \frac{Y}{X} = \frac{\Omega}{(w_x - w_y)}$$



Excitation

- ❑ Piezoelectric excitation by electrodes on the stem

Detection

- ❑ Electrical charges collected on each blade

Coriolis Acceleration

Coriolis coupling FEM analysis with Oofelie

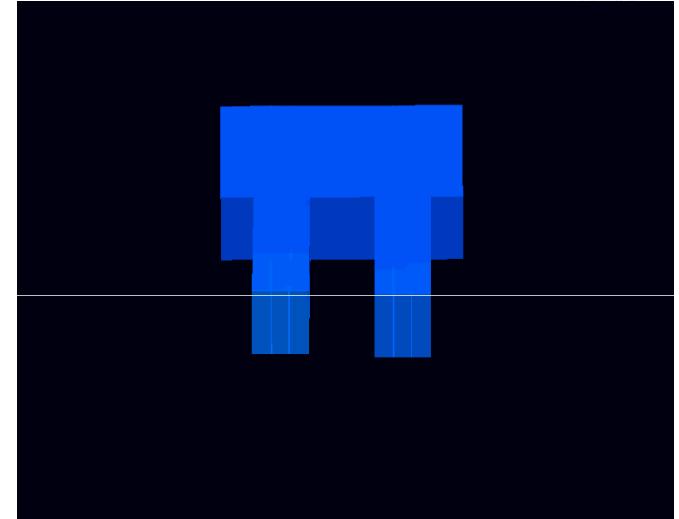
- Harmonic response analysis
- Complex modal analysis

Driving mode excitation by piezoelectricity

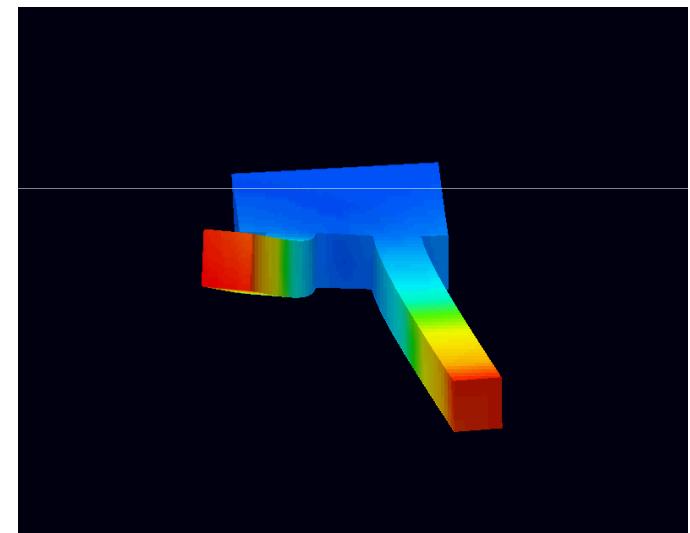
- Electric potentiel : 1V
- Frequency : # 35 kHz
- Driving amplitude : $\sim 1 \mu\text{m}$

Orthogonal vibration due to Coriolis acceleration

- Angular rate : 10 °/s
- Sensing amplitude : 0.2 nm



Driving mode



Coriolis coupling

Design Analysis

Evaluation of total electrical charges

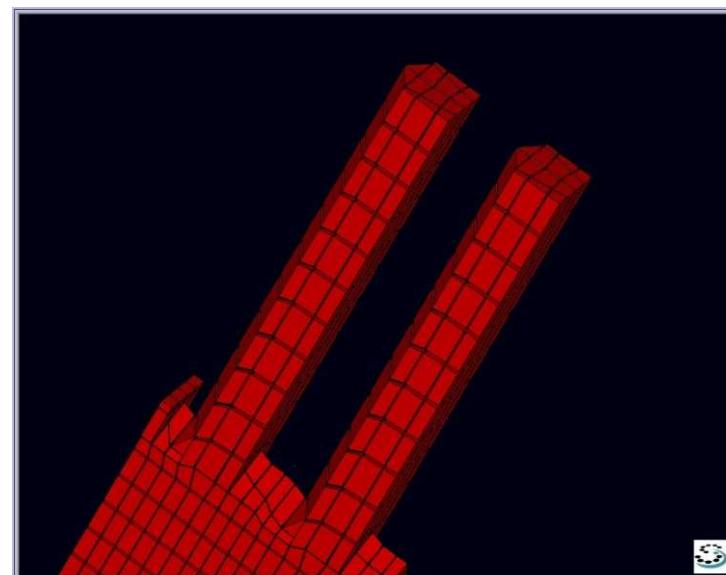
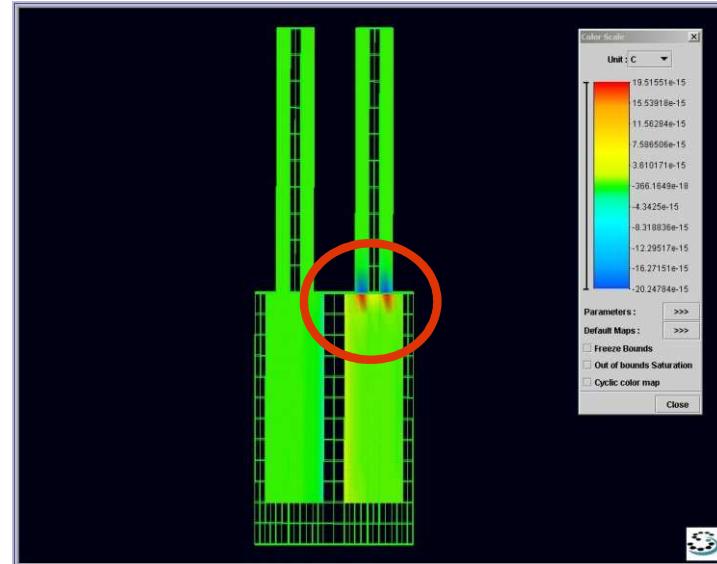
Capacitive coupling

Influence of dissymmetry due to technological processing

- Electrodes misalignment
- Anisotropic chemical etching

Electrodes optimization

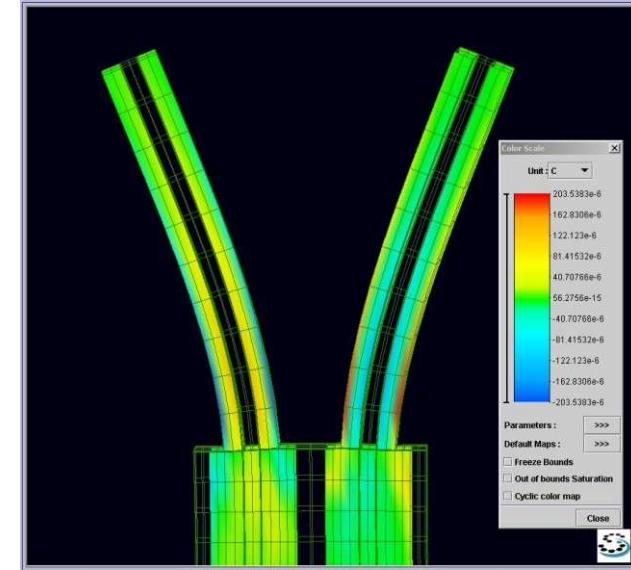
Better understanding of the transducer behavior



Electrode design

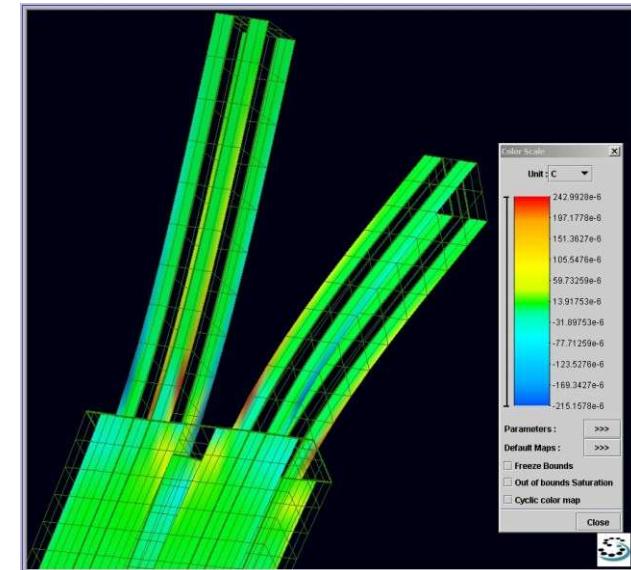
Electrode optimization

- ❑ Piezoelectric modal analysis
- ❑ Electric charge evaluation
- ❑ Electrode efficiency
 - ❑ Mode pilote
 - ❑ Mode Détecteur
- ❑ Optimization of R_m for each mode



Scale factor

- ❑ Harmonic analysis with rotation speed
- ❑ Numeric Scale factor:
 $1.3 \cdot 10^{-16} \text{ C} / (\text{°}/\text{s})$
- ❑ Experimental scale factor:
 $1.6 \cdot 10^{-16} \text{ C} / (\text{°}/\text{s})$
- ❑ Good agreement



Conclusion

❑ FEM analysis for inertial micro-sensors with Oofelie

❑ Multiphysics approach

- ❑ Mechanical
- ❑ Electrical
- ❑ Thermal

❑ Prediction of the main sensor characteristics

- ❑ Quality factor (Thermo-elastic damping)
- ❑ Accelerometer scale factor (Pre-stress analysis)
- ❑ Gyro scale factor (Coriolis coupling)
- ❑ Electric parameters

❑ Good agreement between numeric and experimental results

❑ Development of accurate inertial microsensors

- ❑ Investigation on new materials (GaPO_4) and new designs